

**8-1.** A spherical gas tank has an inner radius of  $r = 1.5$  m. If it is subjected to an internal pressure of  $p = 300$  kPa, determine its required thickness if the maximum normal stress is not to exceed 12 MPa.

$$\sigma_{\text{allow}} = \frac{p r}{2 t}; \quad 12(10^6) = \frac{300(10^3)(1.5)}{2 t}$$

$$t = 0.0188 \text{ m} = 18.8 \text{ mm}$$

**Ans.**

**Ans:**  
 $t = 18.8 \text{ mm}$

**8-2.** A pressurized spherical tank is to be made of 0.5-in.-thick steel. If it is subjected to an internal pressure of  $p = 200$  psi, determine its outer radius if the maximum normal stress is not to exceed 15 ksi.

$$\sigma_{\text{allow}} = \frac{p r}{2 t}; \quad 15(10^3) = \frac{200 r_i}{2(0.5)}$$

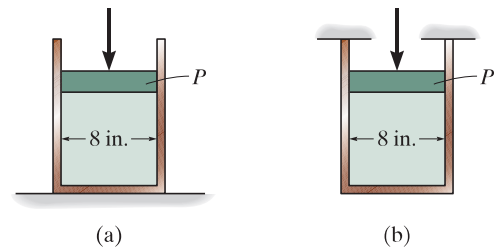
$$r_i = 75 \text{ in.}$$

$$r_o = 75 \text{ in.} + 0.5 \text{ in.} = 75.5 \text{ in.}$$

**Ans.**

**Ans:**  
 $r_o = 75.5 \text{ in.}$

**8-3.** The thin-walled cylinder can be supported in one of two ways as shown. Determine the state of stress in the wall of the cylinder for both cases if the piston  $P$  causes the internal pressure to be 65 psi. The wall has a thickness of 0.25 in. and the inner diameter of the cylinder is 8 in.



Case (a):

$$\sigma_1 = \frac{pr}{t}; \quad \sigma_1 = \frac{65(4)}{0.25} = 1.04 \text{ ksi}$$

**Ans.**

$$\sigma_2 = 0$$

**Ans.**

Case (b):

$$\sigma_1 = \frac{pr}{t}; \quad \sigma_1 = \frac{65(4)}{0.25} = 1.04 \text{ ksi}$$

**Ans.**

$$\sigma_2 = \frac{pr}{2t}; \quad \sigma_2 = \frac{65(4)}{2(0.25)} = 520 \text{ psi}$$

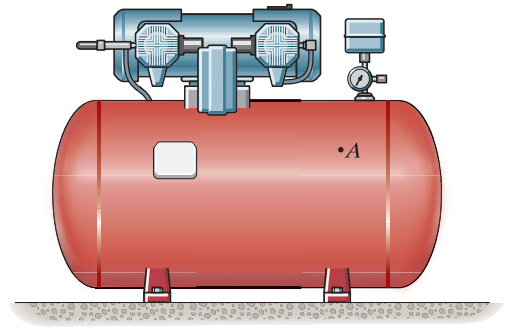
**Ans.**

**Ans:**

(a)  $\sigma_1 = 1.04 \text{ ksi}$ ,  $\sigma_2 = 0$ ,

(b)  $\sigma_1 = 1.04 \text{ ksi}$ ,  $\sigma_2 = 520 \text{ psi}$

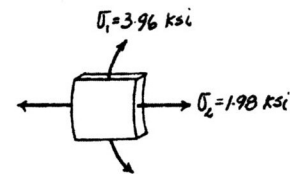
**\*8-4.** The tank of the air compressor is subjected to an internal pressure of 90 psi. If the internal diameter of the tank is 22 in., and the wall thickness is 0.25 in., determine the stress components acting at point *A*. Draw a volume element of the material at this point, and show the results on the element.



**Hoop Stress for Cylindrical Vessels:** Since  $\frac{r}{t} = \frac{11}{0.25} = 44 > 10$ , then *thin wall* analysis can be used. Applying Eq. 8-1

$$\sigma_1 = \frac{pr}{t} = \frac{90(11)}{0.25} = 3960 \text{ psi} = 3.96 \text{ ksi}$$

**Ans.**

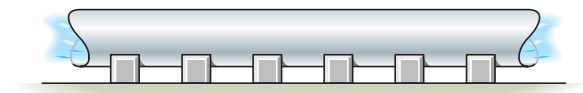


**Longitudinal Stress for Cylindrical Vessels:** Applying Eq. 8-2

$$\sigma_2 = \frac{pr}{2t} = \frac{90(11)}{2(0.25)} = 1980 \text{ psi} = 1.98 \text{ ksi}$$

**Ans.**

**8-5.** The open-ended polyvinyl chloride pipe has an inner diameter of 4 in. and thickness of 0.2 in. If it carries flowing water at 60 psi pressure, determine the state of stress in the walls of the pipe.



$$\sigma_1 = \frac{p r}{t} = \frac{60(2)}{0.2} = 600 \text{ psi}$$

**Ans.**

$$\sigma_2 = 0$$

**Ans.**

There is no stress component in the longitudinal direction since the pipe has open ends.

**Ans:**

$$\sigma_1 = 600 \text{ psi}, \sigma_2 = 0$$