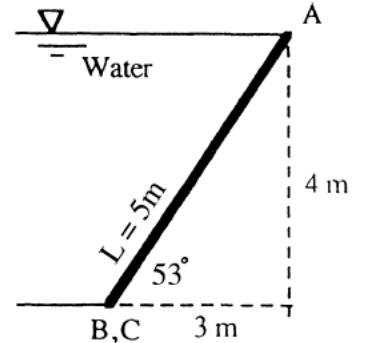


## Problems on Forces on surfaces under fluid, Buoyant force, Stability, and Relative equilibrium

1 :

Panel ABC in the slanted side of a water tank (shown at right) is an isosceles triangle with vertex at A and base BC = 2 m. Find the water force on the panel and its line of action.

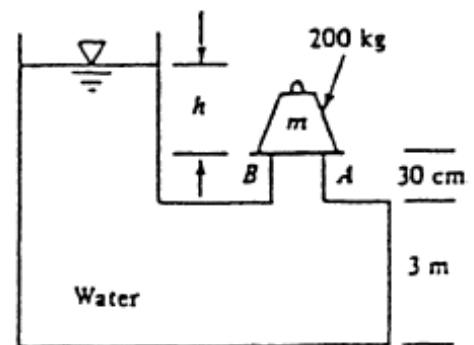


Ans: a-  $F=131000 \text{ N}$ ,

b- The center of pressure is 3.75 m down from A, or 1.25 m up from BC.

2:

In Fig. weightless cover gate AB closes a circular opening 80 cm in diameter when weighed down by the 200-kg mass shown. What water level  $h$  will dislodge the gate?

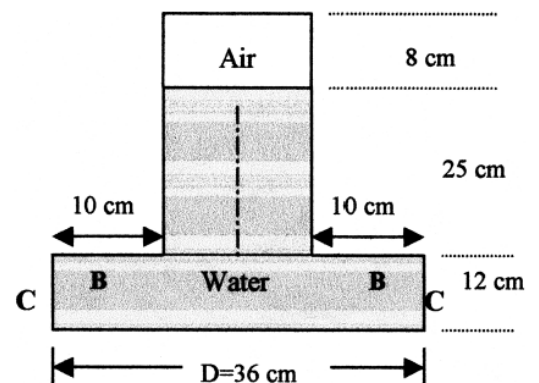


Ans:  $h = 0.40 \text{ m}$

3.

The pressure in the air gap is 8000 Pa gage. The tank is cylindrical. Calculate the net hydrostatic force (a) on the bottom of the tank; (b) on the cylindrical sidewall CC; and (c) on the annular plane panel BB.

Ans:



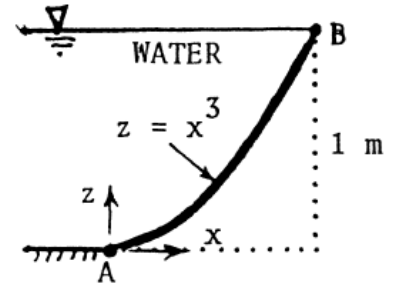
(a) 1180 N.

(b) The net force on the cylindrical sidewall CC= zero due to symmetry.

(c) 853 N

4:

Determine (a) the total hydrostatic force on curved surface AB in Fig. and (b) its line of action. Neglect atmospheric pressure and assume unit width into the paper.



Ans: (a) **8825 N** acting at **56.31°** down and to the right

(b) The line of action of F strikes the vertical above point A at 0.933 m above A, or 0.067 m below the water surface.

Solved problem:1

**2.113** A *spar buoy* is a rod weighted to float vertically, as in Fig. P2.113. Let the buoy be maple wood (SG = 0.6), 2 in by 2 in by 10 ft, floating in seawater (SG = 1.025). How many pounds of steel (SG = 7.85) should be added at the bottom so that  $h = 18$  in?

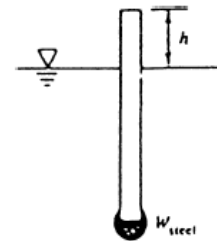


Fig. P2.113

**Solution:** The relevant volumes needed are

$$\text{Spar volume} = \frac{2}{12} \left( \frac{2}{12} \right) (10) = 0.278 \text{ ft}^3; \quad \text{Steel volume} = \frac{W_{\text{steel}}}{7.85(62.4)}$$

$$\text{Immersed spar volume} = \frac{2}{12} \left( \frac{2}{12} \right) (8.5) = 0.236 \text{ ft}^3$$

The vertical force balance is: buoyancy  $B = W_{\text{wood}} + W_{\text{steel}}$ ,

$$\text{or: } 1.025(62.4) \left[ 0.236 + \frac{W_{\text{steel}}}{7.85(62.4)} \right] = 0.6(62.4)(0.278) + W_{\text{steel}}$$

$$\text{or: } 15.09 + 0.1306W_{\text{steel}} = 10.40 + W_{\text{steel}}, \quad \text{solve for } W_{\text{steel}} \approx \mathbf{5.4 \text{ lbf}} \quad \text{Ans.}$$

2:

**2.112** The uniform 5-m-long wooden rod in the figure is tied to the bottom by a string. Determine (a) the string tension; and (b) the specific gravity of the wood. Is it also possible to determine the inclination angle  $\theta$ ?

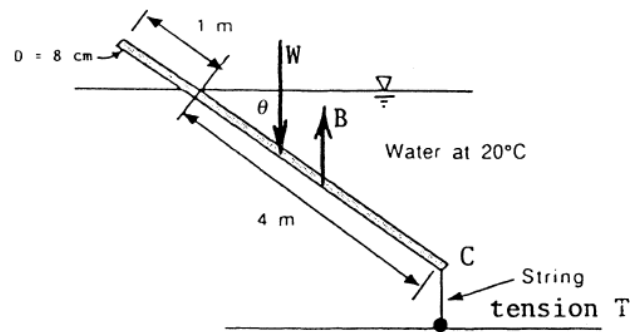


Fig. P2.112

**Solution:** The rod weight acts at the middle, 2.5 m from point C, while the buoyancy is 2 m from C. Summing moments about C gives

$$\sum M_C = 0 = W(2.5 \sin \theta) - B(2.0 \sin \theta), \quad \text{or} \quad W = 0.8B$$

$$\text{But } B = (9790)(\pi/4)(0.08 \text{ m})^2(4 \text{ m}) = 196.8 \text{ N.}$$

$$\text{Thus } W = 0.8B = 157.5 \text{ N} = SG(9790)(\pi/4)(0.08)^2(5 \text{ m}), \quad \text{or: } SG \approx \mathbf{0.64} \quad \text{Ans. (b)}$$

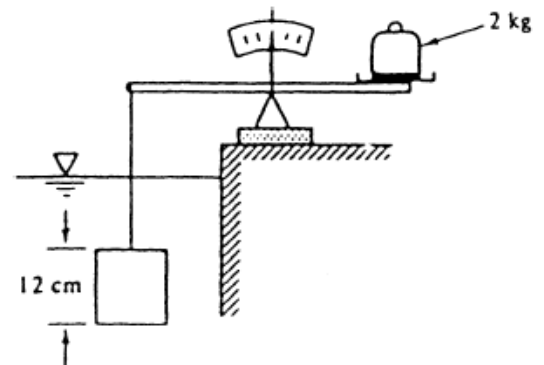
Summation of vertical forces yields

$$\text{String tension } T = B - W = 196.8 - 157.5 \approx \mathbf{39 \text{ N}} \quad \text{Ans. (a)}$$

These results are independent of the angle  $\theta$ , which cancels out of the moment balance.

Pr 5:

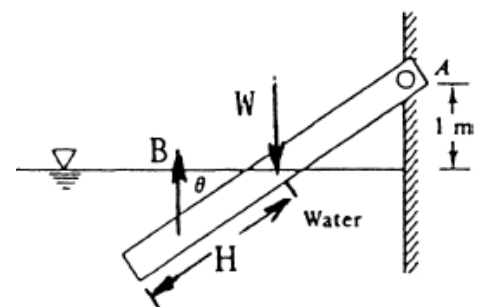
When the 12-cm cube in the figure is immersed in 20°C ethanol, it is balanced on the beam scale by a 2-kg mass. What is the specific gravity of the cube?



Ans:  $\mathbf{19100 \text{ N/m}^3}$

6:

A uniform wooden beam (SG = 0.65) is 10 cm by 10 cm by 3 m and hinged at A. At what angle will the beam float in 20°C water?



$$\gamma_{\text{water}} = 9790 \text{ N/m}^3$$

Ans:  $\theta = 34.3^\circ$

Ex 2:

**2.119** With a 5-lbf-weight placed at one end, the uniform wooden beam in the figure floats at an angle  $\theta$  with its upper right corner at the surface. Determine (a)  $\theta$ ; (b)  $\gamma_{\text{wood}}$ .

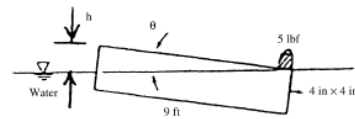


Fig. P2.119

**Solution:** The total wood volume is  $(4/12)^2(9) = 1 \text{ ft}^3$ . The exposed distance  $h = 9 \tan \theta$ . The vertical forces are

$$\sum F_z = 0 = (62.4)(1.0) - (62.4)(h/2)(9)(4/12) - (SG)(62.4)(1.0) - 5 \text{ lbf}$$

The moments of these forces about point C at the right corner are:

$$\sum M_C = 0 = \gamma(1)(4.5) - \gamma(1.5h)(6 \text{ ft}) - (SG)(\gamma)(1)(4.5 \text{ ft}) + (5 \text{ lbf})(0 \text{ ft})$$

where  $\gamma = 62.4 \text{ lbf/ft}^3$  is the specific weight of water. Clean these two equations up:

$$1.5h = 1 - SG - 5/\gamma \quad (\text{forces}) \quad 2.0h = 1 - SG \quad (\text{moments})$$

Solve simultaneously for  $SG \approx \mathbf{0.68}$  Ans. (b);  $h = 0.16 \text{ ft}$ ;  $\theta \approx \mathbf{1.02^\circ}$  Ans. (a)

Ex 3:

**2.134** When floating in water ( $SG = 1$ ), an equilateral triangular body ( $SG = 0.9$ ) might take *two* positions, as shown at right. Which position is more stable? Assume large body width into the paper.

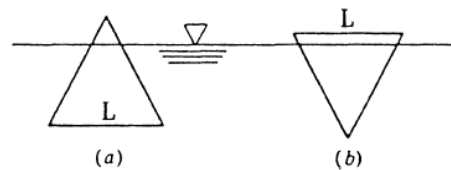


Fig. P2.134

**Solution:** The calculations are similar to the floating cone of Prob. 2.132. Let the triangle be  $L$  by  $L$  by  $L$ . List the basic results.

(a) Floating with point *up*: Centroid  $G$  is  $0.289L$  above the bottom line, center of buoyancy  $B$  is  $0.245L$  above the bottom, hence  $GB = (0.289 - 0.245)L \approx 0.044L$ . Equation (2.52) gives

$$MB = I_o/v_{\text{sub}} = 0.0068L = MG + GB = MG + 0.044L$$

$$\text{Hence } MG = \mathbf{-0.037L} \quad \text{Unstable} \quad \text{Ans. (a)}$$

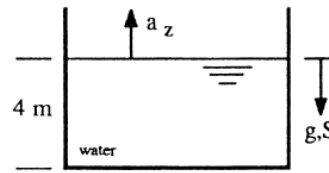
(b) Floating with point *down*: Centroid  $G$  is  $0.577L$  above the bottom point, center of buoyancy  $B$  is  $0.548L$  above the bottom point, hence  $GB = (0.577 - 0.548)L \approx 0.0296L$ . Equation (2.52) gives

$$MB = I_o/v_{\text{sub}} = 0.1826L = MG + GB = MG + 0.0296L$$

$$\text{Hence } MG = \mathbf{+0.153L} \quad \text{Stable} \quad \text{Ans. (b)}$$

Ex 4:

**2.137** A tank of water 4 m deep receives a constant upward acceleration  $a_z$ . Determine (a) the gage pressure at the tank bottom if  $a_z = 5 \text{ m}^2/\text{s}$ ; and (b) the value of  $a_z$  which causes the gage pressure at the tank bottom to be 1 atm.



**Solution:** Equation (2.53) states that  $\nabla p = \rho(\mathbf{g} - \mathbf{a}) = \rho(-k\mathbf{g} - k\mathbf{a}_z)$  for this case. Then, for part (a),

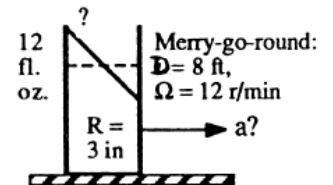
$$\Delta p = \rho(g + a_z)\Delta S = (998 \text{ kg/m}^3)(9.81 + 5 \text{ m}^2/\text{s})(4 \text{ m}) = \mathbf{59100 \text{ Pa (gage)}} \quad \text{Ans. (a)}$$

For part (b), we know  $\Delta p = 1 \text{ atm}$  but we don't know the acceleration:

$$\Delta p = \rho(g + a_z)\Delta S = (998)(9.81 + a_z)(4.0) = 101350 \text{ Pa} \quad \text{if } \mathbf{a_z = 15.6 \frac{m}{s^2}} \quad \text{Ans. (b)}$$

Ex 5:

**2.138** A 12 fluid ounce glass, 3 inches in diameter, sits on the edge of a merry-go-round 8 ft in diameter, rotating at 12 r/min. How full can the glass be before it spills?



**Solution:** First, how high is the container? Well, 1 fluid oz. =  $1.805 \text{ in}^3$ , hence 12 fl. oz. =  $21.66 \text{ in}^3 = \pi(1.5 \text{ in})^2 h$ , or  $h \approx 3.06 \text{ in}$ —It is a fat, nearly square little glass. Second, determine the acceleration toward the center of the merry-go-round, noting that the angular velocity is  $\Omega = (12 \text{ rev/min})(1 \text{ min}/60 \text{ s})(2\pi \text{ rad/rev}) = 1.26 \text{ rad/s}$ . Then, for  $r = 4 \text{ ft}$ ,

$$a_x = \Omega^2 r = (1.26 \text{ rad/s})^2 (4 \text{ ft}) = 6.32 \text{ ft/s}^2$$

Then, for steady rotation, the water surface in the glass will slope at the angle

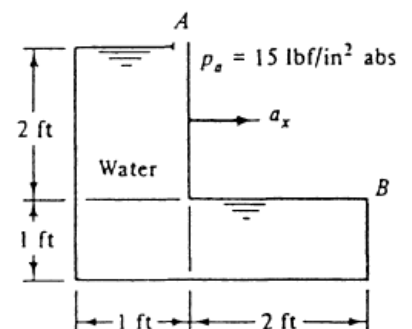
$$\tan \theta = \frac{a_x}{g + a_z} = \frac{6.32}{32.2 + 0} = 0.196, \quad \text{or: } \Delta h_{\text{left to center}} = (0.196)(1.5 \text{ in}) = 0.294 \text{ in}$$

Thus the glass should be filled to no more than  $3.06 - 0.294 \approx 2.77 \text{ inches}$

This amount of liquid is  $v = \pi(1.5 \text{ in})^2(2.77 \text{ in}) = 19.6 \text{ in}^3 \approx \mathbf{10.8 \text{ fluid oz.}}$  Ans.

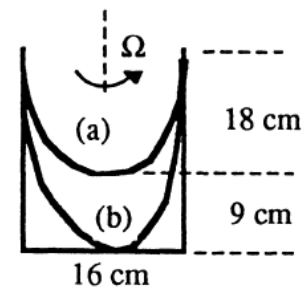
Pr 7:

The tank of water in Fig. .143 is full and open to the atmosphere ( $p_{\text{atm}} = 15 \text{ psi} = 2160 \text{ psf}$ ) at point A, as shown. For what acceleration  $a_x$ , in  $\text{ft/s}^2$ , will the pressure at point B in the figure be (a) atmospheric; and (b) zero absolute (neglecting cavitation)?



Ex 6:

**2.152** A 16-cm-diameter open cylinder 27 cm high is full of water. Find the central rigid-body rotation rate for which (a) one-third of the water will spill out; and (b) the bottom center of the can will be exposed.



**Solution:** (a) One-third will spill out if the resulting paraboloid surface is 18 cm deep:

$$h = 0.18 \text{ m} = \frac{\Omega^2 R^2}{2g} = \frac{\Omega^2 (0.08 \text{ m})^2}{2(9.81)}, \text{ solve for } \Omega^2 = 552,$$

$$\Omega = 23.5 \text{ rad/s} = \mathbf{224 \text{ r/min}} \text{ Ans. (a)}$$

(b) The bottom is barely exposed if the paraboloid surface is 27 cm deep:

$$h = 0.27 \text{ m} = \frac{\Omega^2 (0.08 \text{ m})^2}{2(9.81)}, \text{ solve for } \Omega = 28.8 \text{ rad/s} = \mathbf{275 \text{ r/min}} \text{ Ans. (b)}$$

Ex 7:

**2.155** For what uniform rotation rate in r/min about axis C will the U-tube fluid in Fig. P2.155 take the position shown? The fluid is mercury at 20°C.

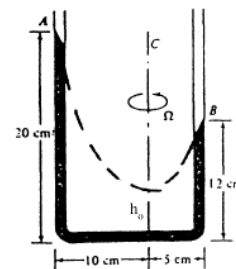


Fig. P2.155

**Solution:** Let  $h_o$  be the height of the free surface at the centerline. Then, from Eq. (2.64),

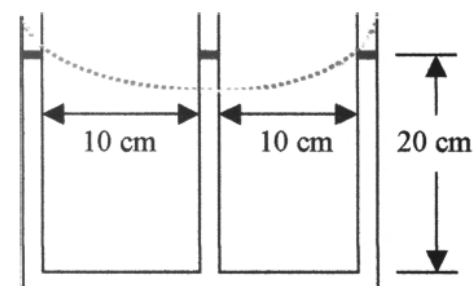
$$z_B = h_o + \frac{\Omega^2 R_B^2}{2g}; \quad z_A = h_o + \frac{\Omega^2 R_A^2}{2g}; \quad R_B = 0.05 \text{ m} \quad \text{and} \quad R_A = 0.1 \text{ m}$$

$$\text{Subtract: } z_A - z_B = 0.08 \text{ m} = \frac{\Omega^2}{2(9.81)} [(0.1)^2 - (0.05)^2],$$

$$\text{solve } \Omega = 14.5 \frac{\text{rad}}{\text{s}} = \mathbf{138 \frac{\text{r}}{\text{min}}} \text{ Ans.}$$

Pr 8:

The three-legged manometer in Fig. ( ) is filled with water to a depth of 20 cm. All tubes are long and have equal small diameters. If the system spins at angular velocity  $\Omega$  about the central tube, (a) derive a formula to find the change of height in the tubes; (b) find the height in cm in each tube if  $\Omega = 120 \text{ rev/min}$ . [HINT: The central tube must supply water to both the outer legs.]

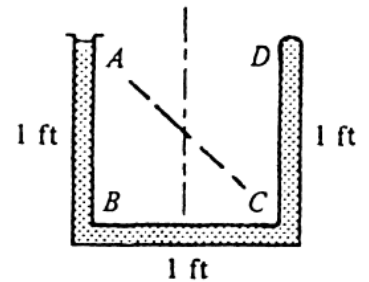


Ans: (a)  $-\frac{\Omega^2 R^2}{3g}$ , (b)  $-0.054 \text{ m (down)}$

Pr 9:

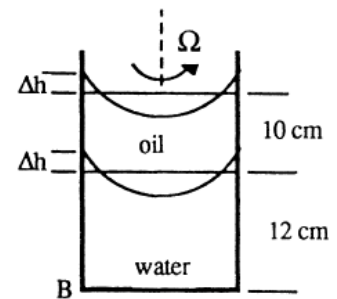
----- The U-tube in Fig. is open at A and closed at D. What uniform acceleration  $a_x$  will cause the pressure at point C to be atmospheric? The fluid is water.

Ans:  $32.2 \text{ ft/s}^2$



Pr 10:

A very deep 18-cm-diameter can has 12 cm of water, overlaid with 10 cm of SAE 30 oil. It is rotated about the center in rigid-body motion at 150 r/min. (a) What will be the shapes of the interfaces? (b) What and where will be the maximum fluid pressure?



Ans: (a)

same-shape paraboloid, with a deflection  $\Delta h$  up at the wall and down in the center = 5.1 cm

(b) 2550 Pa.